# Solving the elastic problem in Finite Elements

$$\frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \sigma_{xy}}{\partial y} = 0$$
$$\frac{\partial \sigma_{xy}}{\partial x} + \frac{\partial \sigma_{yy}}{\partial y} = 0,$$

Non divergence of stresses:

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Stress-strain relation:

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{bmatrix} = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & 0 \\ \nu & 1-\nu & 0 \\ 0 & 0 & \frac{1}{2}(1-2\nu) \end{bmatrix} \begin{bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \gamma_{xy} \end{bmatrix}$$

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Strain-displacement relation: (u is not a velocity!)

$$\begin{bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \gamma_{xy} \end{bmatrix} = \begin{bmatrix} \frac{\partial}{\partial x} & 0 \\ 0 & \frac{\partial}{\partial y} \\ \frac{\partial}{\partial y} & \frac{\partial}{\partial x} \end{bmatrix} \begin{bmatrix} u_x \\ u_y \end{bmatrix}$$

$$egin{aligned} \mathbf{B}^T \hat{oldsymbol{\sigma}} &= \mathbf{0}, \ \hat{oldsymbol{\sigma}} &= \mathbf{D} \hat{oldsymbol{\epsilon}}, \ \hat{oldsymbol{\epsilon}} &= \mathbf{B} \mathbf{e}, \end{aligned}$$

 $\partial$  $\mathbf{B}^T \hat{\boldsymbol{\sigma}} = \mathbf{0},$  $\mathbf{B} = \begin{vmatrix} \frac{\partial}{\partial x} & 0\\ 0 & \frac{\partial}{\partial y}\\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} \end{vmatrix}$  $\hat{\boldsymbol{\sigma}} = \mathbf{D}\hat{\boldsymbol{\epsilon}}, \ \hat{\boldsymbol{\epsilon}} = \mathbf{B}\mathbf{e},$ 

**Differential operator** 

 $\overline{\partial x}$ 





$$\mathbf{B}^{T} \hat{\boldsymbol{\sigma}} = \mathbf{0}, \\ \hat{\boldsymbol{\sigma}} = \mathbf{D} \hat{\boldsymbol{\epsilon}}, \\ \hat{\boldsymbol{\epsilon}} = \mathbf{B} \mathbf{e}, \end{bmatrix} \mathbf{B} = \begin{bmatrix} \frac{\partial}{\partial x} & 0 \\ 0 & \frac{\partial}{\partial y} \\ \frac{\partial}{\partial y} & \frac{\partial}{\partial x} \end{bmatrix} \quad \hat{\boldsymbol{\sigma}} = \begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{bmatrix}$$
Stress-strain operator
$$\mathbf{D} = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & 0 \\ \nu & 1-\nu & 0 \\ 0 & 0 & \frac{1}{2}(1-2\nu) \end{bmatrix}$$

Strain components

$$\hat{oldsymbol{\epsilon}} = egin{bmatrix} \epsilon_{xx} \ \epsilon_{yy} \ \gamma_{xy} \end{bmatrix}$$



Strain components

 $\hat{\boldsymbol{\epsilon}} = \begin{bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \gamma_{xy} \end{bmatrix} \qquad \mathbf{e} = \begin{bmatrix} u_x \\ u_y \end{bmatrix}$ 

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Stress-strain operator
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Strain components

 $\hat{oldsymbol{\epsilon}} = egin{bmatrix} \epsilon_{xx} \ \epsilon_{yy} \ \gamma_{xy} \end{bmatrix}$ 

Displacement components

$$\mathbf{e} = \begin{bmatrix} u_x \\ u_y \end{bmatrix}$$

Result:  $\mathbf{B}^T \hat{\boldsymbol{\sigma}} = \mathbf{0}$   $\mathbf{B}^T \mathbf{D} \hat{\boldsymbol{\epsilon}} = \mathbf{0}$  $\mathbf{B}^T \mathbf{D} \mathbf{B} \mathbf{e} = \mathbf{0}$   $\mathbf{B}^{T} \hat{\boldsymbol{\sigma}} = \mathbf{0}$  $\mathbf{B}^{T} \mathbf{D} \hat{\boldsymbol{\epsilon}} = \mathbf{0}$  $\mathbf{e} = \begin{bmatrix} u_{x} \\ u_{y} \end{bmatrix}$  $\mathbf{B}^{T} \mathbf{D} \mathbf{B} \mathbf{e} = \mathbf{0}$ 

 $\mathbf{B}^T \hat{\boldsymbol{\sigma}} = \mathbf{0}$  $\mathbf{B}^T \mathbf{D} \hat{\boldsymbol{\epsilon}} = \mathbf{0}$  $\mathbf{e} = \begin{vmatrix} u_x \\ u_y \end{vmatrix}$  $\mathbf{B}^T \mathbf{D} \mathbf{B} \mathbf{e} = \mathbf{0}$ 

Weak form?

$$\begin{split} \mathbf{B}^T \hat{\boldsymbol{\sigma}} &= \mathbf{0} \\ \mathbf{B}^T \mathbf{D} \hat{\boldsymbol{\epsilon}} &= \mathbf{0} \\ \mathbf{B}^T \mathbf{D} \mathbf{B} \mathbf{e} &= \mathbf{0} \\ \end{split} \\ \mathbf{B}^T \mathbf{D} \mathbf{B} \mathbf{e} &= \mathbf{0} \\ \mathbf{W} eak \text{ form?} \\ u_x(x,y) &\approx \begin{bmatrix} N_1(x,y) & N_2(x,y) & N_3(x,y) & N_4(x,y) \end{bmatrix} \begin{bmatrix} u_x(1) \\ u_x(2) \\ u_x(3) \\ u_x(4) \end{bmatrix} = \mathbf{N} \mathbf{u}_x \\ u_y(x,y) &\approx \begin{bmatrix} N_1(x,y) & N_2(x,y) & N_3(x,y) & N_4(x,y) \end{bmatrix} \begin{bmatrix} u_y(1) \\ u_y(2) \\ u_y(3) \\ u_y(4) \end{bmatrix} = \mathbf{N} \mathbf{u}_y \end{split}$$

Displacement vector in weak form:

$$\mathbf{r} = egin{bmatrix} u_x \ u_y \end{bmatrix} = egin{bmatrix} u_{x(1)} \ u_{x(2)} \ u_{x(3)} \ u_{x(4)} \ u_{y(1)} \ u_{y(2)} \ u_{y(3)} \ u_{y(4)} \end{bmatrix}$$

## Displacement vector in weak form:



Problem in weak form:

## $\mathbf{K}\mathbf{r} = \mathbf{F}$

#### **Displacement vector** in weak form:



Problem in weak form:

 $\mathbf{K}\mathbf{r} = \mathbf{F}$ 

where  $\mathbf{K} = \int_{\Omega^e} \hat{\mathbf{B}}^T \mathbf{D} \hat{\mathbf{B}} \, dV,$ 

## Displacement vector in weak form:



#### **Displacement vector** in weak form:



#### Global node and element numbering



 $\begin{array}{cccccccc} g\_num = \begin{bmatrix} 1 & 2 & 3 & 5 & 6 & 7 & 9 & 10 & 11 \\ 5 & 6 & 7 & 9 & 10 & 11 & 13 & 14 & 15 \\ 6 & 7 & 8 & 10 & 11 & 12 & 14 & 15 & 16 \\ 2 & 3 & 4 & 6 & 7 & 8 & 10 & 11 & 12 \end{bmatrix}$ 

Relationship between elements and global node numbers

nf =  $\begin{bmatrix} 1 \ 3 \ 5 \ 7 \ 9 \end{bmatrix}$  11 13 15 17 19 21 23 25 27 29 31 2 4 6 8 10 12 14 16 18 20 22 24 26 28 30 32

Relationship between nodes and equation numbers

g\_g=

 $\begin{bmatrix} 1 & 3 & 5 & 9 & 11 & 13 & 17 & 19 & 21 \\ 9 & 11 & 13 & 17 & 19 & 21 & 25 & 27 & 29 \\ 11 & 13 & 15 & 19 & 21 & 23 & 27 & 29 & 31 \\ 3 & 5 & 7 & 11 & 13 & 15 & 19 & 21 & 23 \\ 2 & 4 & 6 & 10 & 12 & 14 & 18 & 20 & 22 \\ 10 & 12 & 14 & 18 & 20 & 22 & 26 & 28 & 30 \\ 12 & 14 & 16 & 20 & 22 & 24 & 28 & 30 & 32 \\ 4 & 6 & 8 & 12 & 14 & 16 & 20 & 22 & 24 \end{bmatrix}$ 

Relationship between elements and equation numbers